

Hydrodynamic analysis and numerical modelling of heave plates dedicated to the design of FOWT

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S.Y. HAN¹, B. Bouscasse¹, J.-C. Gilloteaux¹, D. Le Touze[†]

¹Ecole Centrale de Nantes/CNRS



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Introduction

> FLOAWER Project, ESR 5 (WP5-Advanced Floater Analysis)

- **Topic**
Hydrodynamic analysis and numerical modeling of Heave Plates
- **Expected results** (2020 ~ 2023, 3yrs)
 - a. Hydrodynamic database
 - b. Improved engineering models

> Heave plate in Floating Wind Turbine

- Increase **Added Mass**
 - ✓ Escape from resonance period $T_3 = 2\pi \sqrt{\frac{m + m_a}{\rho g A_{wp} + k_m}}$
- Increase **Total Damping**
 - ✓ Reduce the wave induced response



- Improve **Structural Design**
 - ✓ Floater, column and mooring lines
- Improve **Power Output** of Floating Wind Turbine

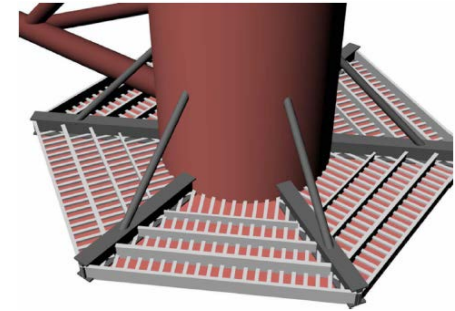


Figure 1. Heave plate on Windfloat (D. Roddier et al., 2010)



a. Windfloat (D. Roddier et al., 2010)



b. Floatgen (Floatgen report #3.1)

Figure 2. Floating offshore wind turbine with the heave plates

Key Features

> Heave plate in Floating Wind Turbine

• Flow Parameters

- ✓ Keulegan Carpenter (KC) Number

$$KC_C = \frac{2\pi\xi_3}{D_C} \text{ or } KC_D = \frac{2\pi\xi_3}{D_D}$$

- Classic Offshore Platform, $0.0 < KC_C < 1.2$

- Floating Offshore Wind Turbine

ex> HiPRWind, $KC_C = 2.9$ in extreme condition

- ✓ Frequency Parameter and Reynolds number

$$\beta_C = \frac{D_C^2 f_3}{\nu} \text{ or } \beta_D = \frac{D_D^2 f_3}{\nu} \text{ and } Re = (KC)\beta$$

• Geometrical Parameters

- ✓ Diameter ratio

$$1.0 < D_D / D_C < 3.6$$

- ✓ Draft ratio

$$0.4 < d / r_D < 25.5$$

where r_D is $\frac{1}{2} D_D$

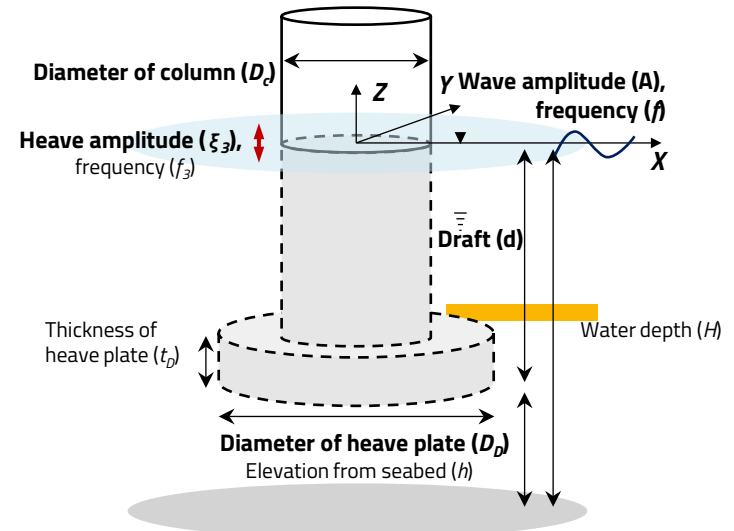


Figure 3. Definition of parameters

Table 1. Floating Platform Models

Type		Semi1	Semi2	Semi3	Semi4	Spar1	Spar2	Barge
Column Dia.	D_C [m]	12.0 ^a	10.5	7.0	15.8	6.5/9.4 ^b	14.4	34.0
Heave Plate Dia.	D_D [m]	24.0	36.9	20.0	22.8	-	-	38.4
Draft	d [m]	20.0	18.0	15.5	22.0	120.0	78.0	7.0
Plate Thickness	t_D [m]	6.0	thin plate	0.1	thin plate	-	-	thin plate
Diameter Ratio	D_D / D_C	2.0	3.6	2.9	1.4	1.0	1.0	1.1
Draft Ratio	h / r_D	1.7	1.2	1.6	1.9	25.5 ^c	10.8 ^d	0.4

^a Offset column, ^b tapered column, ^{c,d} Draft ratio with column diameter

< Note > Semi1: DeepCWind (OC4), Semi2: Windfloat, Semi3: HiPRWind (OC3), Semi4: OO-STAR, Spar1: Hywind, Spar2: Hywind-Scotland, Barge: Floatgen (Idea)

Hydrodynamic Database with Heave Plate

> List of Previous Database

- Forced oscillation in Heave motion

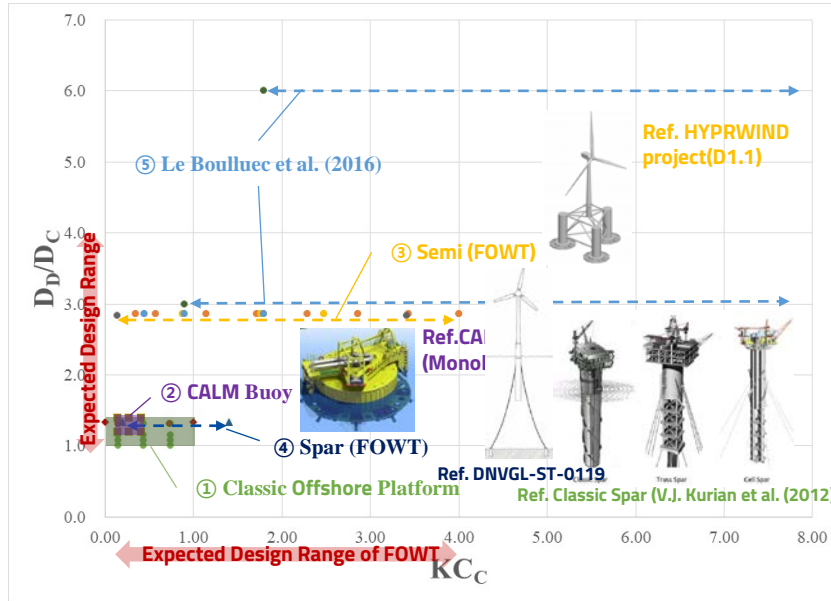
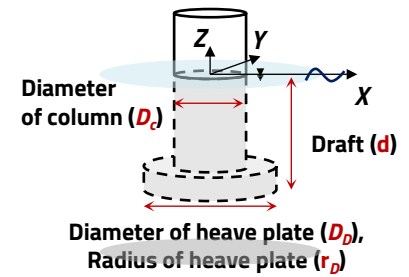


Fig 5. Hydrodynamic database with KC_c vs Diameter ratio

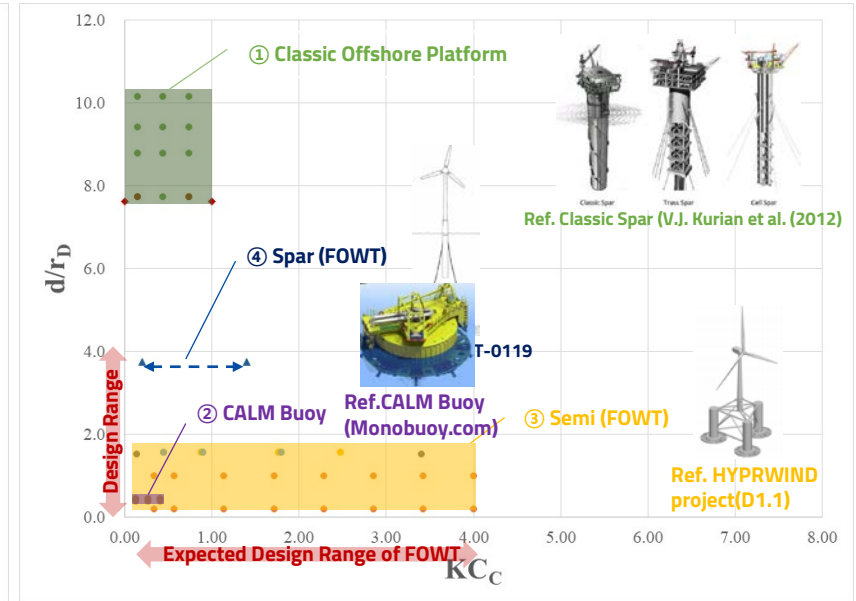


Fig 6. Hydrodynamic database with KC_c vs draft ratio

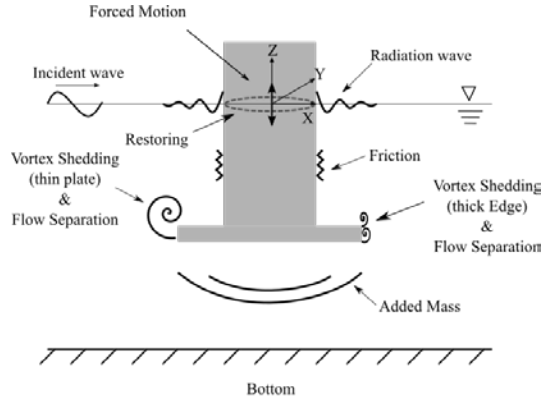
Summary for Future Works

> Methodologies

- Experiment & Numerical Analysis (OpenFOAM)
- Forced oscillation and Captive Test

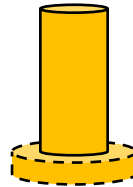
> Test Campaign

1. Forced Oscillation (Radiation Problem)



- Including **free surface** and **wave effect**
- **Complex motion** (regular and irregular)

< Model >



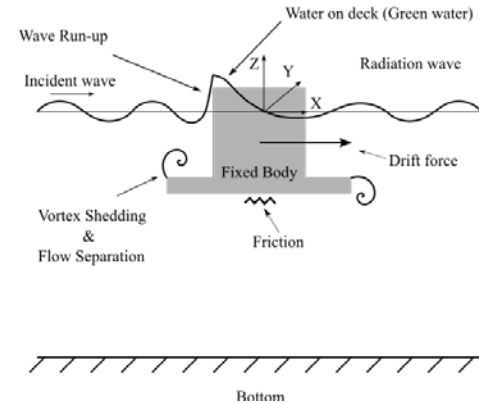
Single Column
with Heave Plate



< Output >

- ✓ Hydrodynamic **D/B**
- ✓ Numerical Scheme

2. Captive (Diffraction Problem)



- **Wave drift force**
- **Wave run-up** (with heave plate)
- Upright & **inclined** conditions

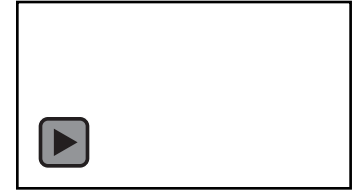


Fig 6. Ocean Engineering Tank (LHEEA) and CFD (OpenFOAM)

Thanks for your attention



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APPENDIX A – Formula for Added Mass Coefficients

Configuration	Section	Added mass (or coefficient)	Ref.
Isolated heave plate	Cir.	$m_a = \frac{1}{3}\rho D_d^3$	[11]
A cylinder with heave plate	Cir.	$C_a = \frac{1}{3} \frac{2D_d^3 + 3\pi D_d^2 z - \pi^3 z^3 - 3\pi D_c^2 z}{\pi(D_d^2 t_d + D_c^2 t_c)},$ $z = \frac{1}{\pi} \sqrt{D_d^2 - D_c^2}$	[19]
	Cir., Oct., Rec.	$C_a = f_{r1} \left(k^3 - \frac{1}{4} \left[3r_d^2 \sqrt{k^2 - r_d^2} + \left(k - \sqrt{k^2 - r_d^2} \right)^2 \left(2k + \sqrt{k^2 - r_d^2} \right) \right] \right),$ $k = k_1 \cdot k_2, \quad k_1 = 1 + 0.2KC, \quad k_2 = \begin{cases} 1.00; & \text{Circular heave plate} \\ 0.95; & \text{Octagonal heave plate} \\ 0.75; & \text{Square heave plate} \end{cases}$ $f_{r1} = \frac{1}{3}\rho D_d^3 / \left(\frac{1}{4}\pi(D_d^2 t_d + D_c^2 t_c) \right)$	[20]
A cylinder with multiple heave plates	Cir.	$C_a = \frac{1}{12} \frac{8D_d^3 - 12\pi D_c^2 z - 12\pi D_c^2 L + 12\pi D_d^2 z - 4\pi^3 z^3 + 12\pi D_d^2 L - \pi^3 L^3}{\pi(2D_d^2 t_d + D_c^2 t_c)}$	[19]
	Cir., Oct., Rec.	$C_a = \begin{cases} f_{r2} \left(2k^3 - \frac{1}{4} \left[3r_d^2 \sqrt{k^2 - r_d^2} + \left(k - \sqrt{k^2 - r_d^2} \right)^2 \left(2k + \sqrt{k^2 - r_d^2} \right) \right] \right. \\ \quad \left. - \frac{1}{16} [12\pi r_d^2 r_L + (2k - \pi r_L)^2 (4k + \pi r_L)] \right); & \text{for } r_L \leq 2k/\pi \\ f_{r2} \left(2k^3 - \frac{3}{4} \left[3r_d^2 \sqrt{k^2 - r_d^2} + \left(k - \sqrt{k^2 - r_d^2} \right)^2 \left(2k + \sqrt{k^2 - r_d^2} \right) \right] \right); & \text{for } r_L > 2k/\pi \end{cases}$ $f_{r2} = \frac{1}{3}\rho D_d^3 / \left(\frac{1}{4}\pi(2D_d^2 t_d + D_c^2 t_c) \right)$	[20]

APPENDIX B – Formula for Drag Coefficients

Configuration	Section	Drag coefficient	Ref.
Isolated heave plate	Cir.	$C_{d(form)} = A(KC)^n$ $A = \begin{cases} 11.8 \\ 5.7 \end{cases}, n = \begin{cases} -1/3 & \text{for flat plate} \\ 0 & \text{for diamond cylinder} \end{cases}$	[23]
A cylinder with heave plate	Cir.	$C_{d(form)} = A(KC)^n$ $A = \begin{cases} 0.15 \\ 2.5 \\ 4.0 \end{cases}, n = \begin{cases} -3/4 & \text{for independent vortex shedding} \\ -1/5 & \text{for interactive vortex shedding} \\ -1/4 & \text{for uni-directional vortex shedding} \end{cases}$	[24]
	Cir., Oct., Rec.	$C_d = \min \{ 1.7r_t^{-1/3.7} (KC)^{-1/k_3}, 12 \}$ $k_3 = \begin{cases} 2.5; & \text{Circular heave plate} \\ 2.5; & \text{Octagonal heave plate} \\ 3.0; & \text{Square heave plate} \end{cases}$	[20]
A cylinder with multiple heave plate	Cir., Oct., Rec.	$C_d = \begin{cases} \min \{ 1.7 (r_{t,d1}^{-1/3.7} + r_{t,d2}^{-1/3.7}) (KC)^{-1/k_3} - 3.7k_2 + 2.9r_L, 24 \}; & \text{for } r_L \leq 2k_2/\pi \\ \min \{ 1.7 (r_{t,d1}^{-1/3.7} + r_{t,d2}^{-1/3.7}) (KC)^{-1/k_3}, 24 \}; & \text{for } r_L > 2k_2/\pi \end{cases}$	[20]

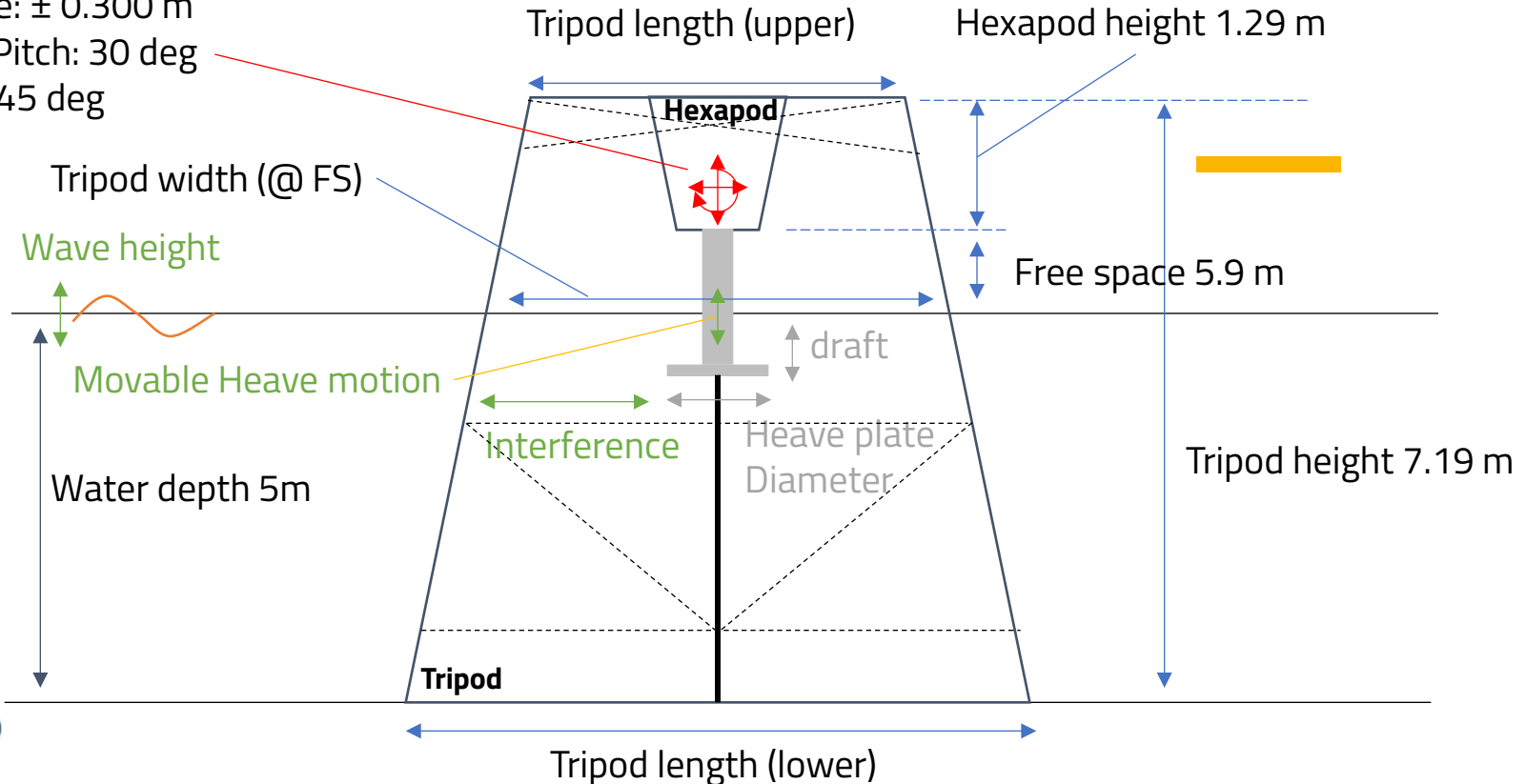
APPENDIX C – Test Set-up

Surge, Sway: ± 0.465 m

Heave: ± 0.300 m

Roll, Pitch: 30 deg

Yaw: 45 deg



APPENDIX D – Time series

From Geometry

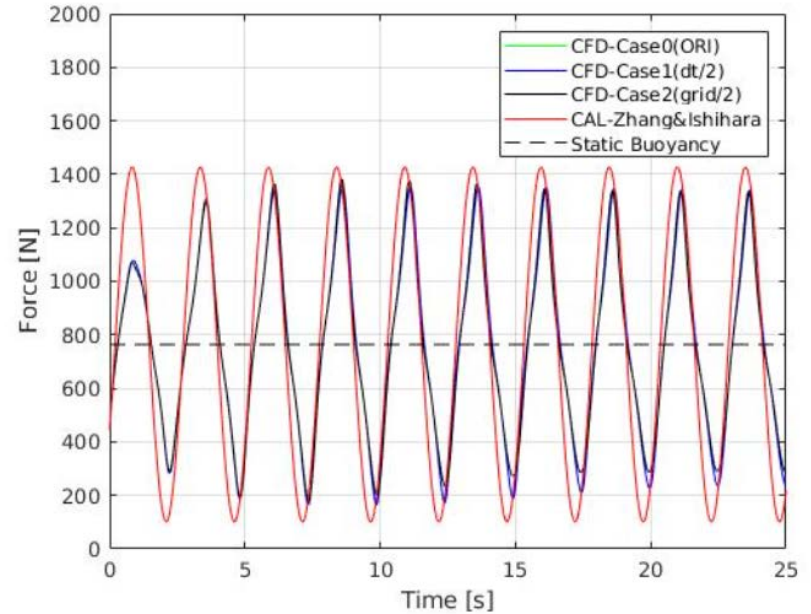
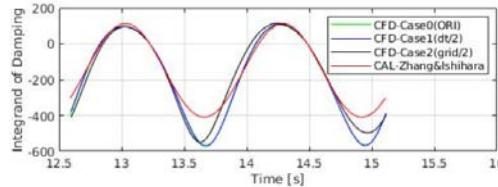
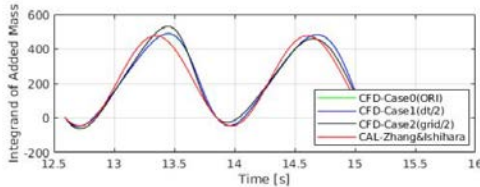
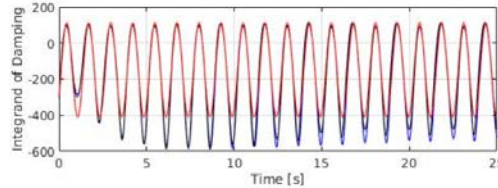
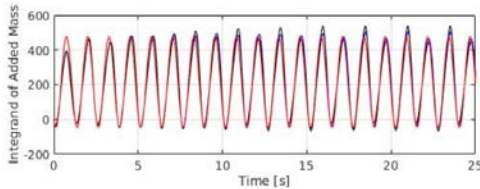
$$F_{measured} = -M_a \ddot{x}(t) - B_{eq} \dot{x}(t) - \rho g A_w x(t) + \rho g \nabla$$

Computed by Zhang & Ishihara formula

< Integrand for Added mass and Damping >

$$\int_t^{t+T} a\omega^2 M_a \sin \omega t \sin \omega t dt - \int_t^{t+T} a\omega B \cos \omega t \sin \omega t dt = \int_t^{t+T} F_{hyd}(t) \sin \omega t dt$$

$$\int_t^{t+T} a\omega^2 M_a \sin \omega t \cos \omega t dt - \int_t^{t+T} a\omega B \cos \omega t \cos \omega t dt = \int_t^{t+T} F_{hyd}(t) \cos \omega t dt$$



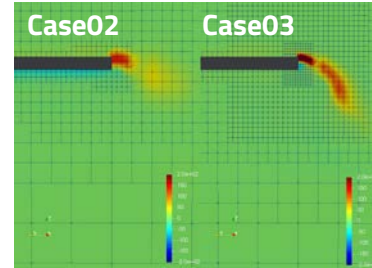
APPENDIX E – Vorticity Fields

< Main Dimensions >

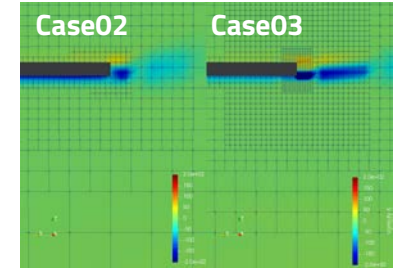
Nomenclature	Unit	Value	Note
Dc	m	0.350	Column diameter
h	m	0.775	Draft of floater
Dd	m	1.000	Disc diameter (heave plate)
td	m	0.005	Disc thickness
Amplitude	m	0.140	Heave amplitude
Period	s	2.519	Heave period
KC	-	0.866	KC number
Beta	-	411738	frequency parameter

< Test Cases for convergence >

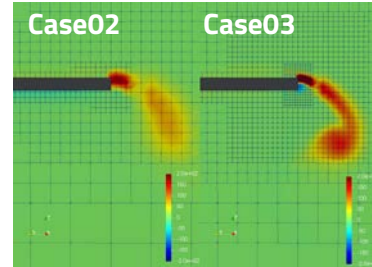
	Case01	Case02	Case03
Turbulence Model	kOmegaSST		
Motion	Moving wall velocity		
Freesurface	Volume of Fraction		
Time Step	0.002	0.001	0.002
Number of Grids	0.8M	0.8M	1.6M



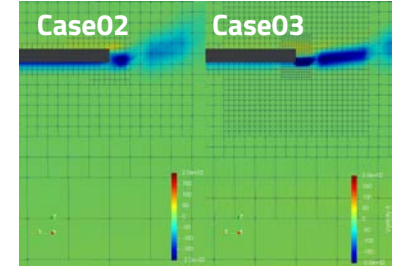
a. $t/T=8.69$



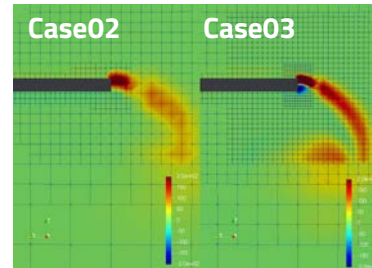
d. $t/T=9.21$



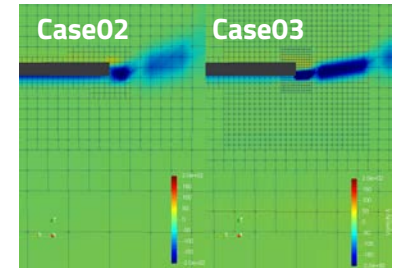
b. $t/T=8.73$



e. $t/T=9.25$



c. $t/T=8.77$



f. $t/T=9.29$

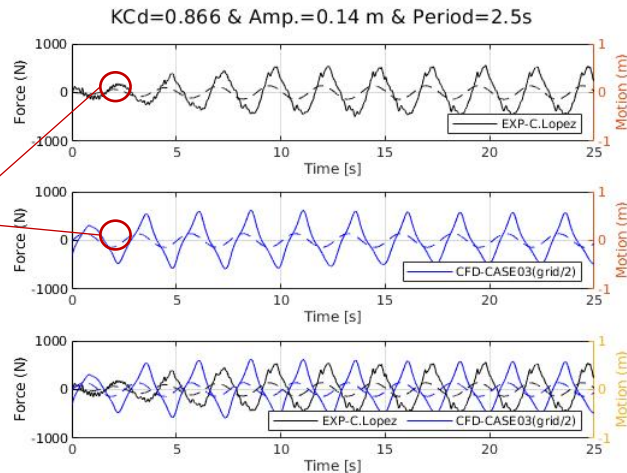
APPENDIX F – Experiment (C.Lopez)

< Main Dimensions >

Nomenclature	Unit	Value	Note
Dc	m	0.350	Column diameter
h	m	0.775	Draft of floater
Dd	m	1.000	Disc diameter (heave plate)
td	m	0.005	Disc thickness
Amplitude	m	0.140	Heave amplitude
Period	s	2.519	Heave period
KC	-	0.866	KC number
Beta	-	411738	frequency parameter

Phase different!

Phase shifting



< Test Cases for convergence >

	Case01	Case02	Case03
Turbulence Model	kOmegaSST		
Motion	Moving wall velocity		
Freesurface	Volume of Fraction		
Time Step	0.002	0.001	0.002
Number of Grids	0.8M	0.8M	1.6M

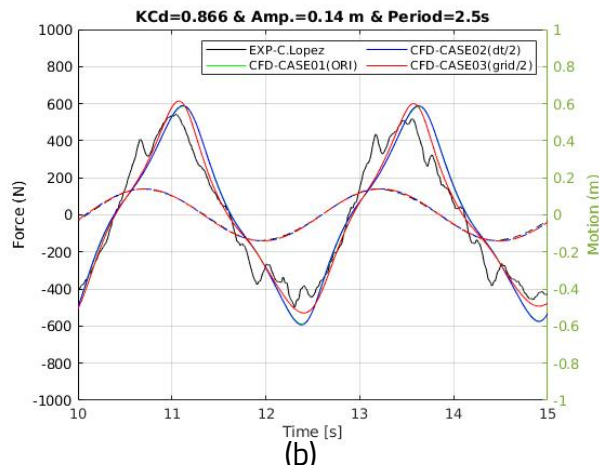
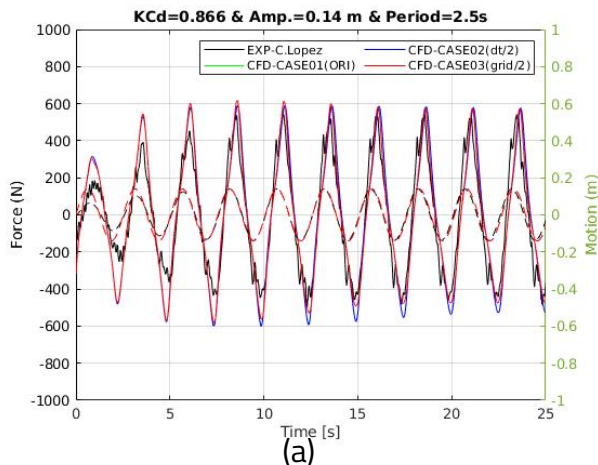


Fig. Force time history comparison (EXP vs CFD) (a) Full time history and (b) Zoom in